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GASKET UNIT FOR A BEARING BUSHING

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FIELD OF THE INVENTION

The invention relates to a gasket unit for a rolling bearing supported journal bearing in a bearing bushing. In particular, the gasket unit is provided for bearing bushings used, for example, in the universal joint of a drive shaft. The components provided here are a reinforced main seal, inserted in a fixed manner in a bore of the bearing bushing, and a front seal positioned axially in front of the main seal and connected to the journal in a torque proof manner, and a spring washer arranged between the main seal and the face of the rollers. For this purpose, the main seal is inserted into the bore of the bearing bushing in a friction-locked manner via a cylindrical section of a reinforcement. In the mounting position at least one sealing lip is linked to a blade of the reinforcement, directed radially inward and supported at the journal in a sealing manner. The front seal covers an annular gap formed between the bearing bushing and the journal in the mounting position. The spring washer, also provided in the gasket unit, is supported on the outside at the reinforcement of the main seal and on the inside at the face of the rolls.

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BACKGROUND

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A gasket unit is known from DE 44 08 831 A1, which, in addition to the above-mentioned individual components, such as main seal, front seal, and spring washer, also comprises an intermediate seal. Said seal is provided with a reinforcement, which is supported at its face on the bearing bushing and with its interior cylindrical blade being extrusion-coated by the

insulating material of the intermediate seal. The exterior cylindrical blade of the reinforcement has a tolerance and engages an annular gap of the front seal. A disadvantage in this known gasket unit is that it enlarges the axial construction space and requires a more complex mounting expense.

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SUMMARY

The present invention is based on the objective of providing a gasket unit with a reduced volume requirement for its components, provided with a reduced wear of the sealing lip and the sealing effect being optimized with respect to the discharge of lubricants and/or the entry of contaminants.

This objective is attained in the main seal being provided with a reinforcement facing with its end in a direction of the opening of the bearing bushing, bent on the outside and supported on the interior wall of the bearing bushings. This measure ensures an advantageously positioned arrangement of the main seal in the bearing bushing. Connected thereto, a defined axial support force of the spring washer develops, which is allocated between the main seal and the rollers of the journal bearing. Furthermore, the mounted position of the main seal according to the invention ensures that the sealing lip corresponding to the main seal is located in a defined mounting position at the outer surface of the journal. According to the invention the main seal comprises two axially distanced sealing lips, sealingly supported concurrently at a section of the journal and having the same diameter. An increased support force of the sealing lip allocated to the front seal is achieved by way of a tubular spring inserted in an exterior circumferential groove of the sealing lip and pressing it to the journal with an enhanced radial force. The exterior of the front seal, arranged in a torque-proof manner on the journal, is located in a form-fitting manner to a radially separated end section of the bearing bushing, with the combination of these components forming a

labyrinth seal. Furthermore, in the mounted position the front seal comprises a sealing lip allocated to the inside of the reinforcement of the main seal.

- 5 The measures according to the invention advantageously reduce the wear of the sealing lips and simultaneously improve the sealing effect of the gasket unit regarding the discharge of lubricants, which directly influences bearing life negatively. The invention is further characterized by an advantageously low axial pre-tension of the rollers due to a relaxation of the elastomer seal, 10 i.e., the material of the main seal. In an advantageous manner, the spring washer according to the invention, arranged between the main seal and the rollers, is almost directly supported on the outside of the reinforcement of the main seal, so that a tightly toleranced axial force and/or front tension of the rolling bearing can develop in connection with the defined mounted position 15 of the main seal. Further, the structure of the gasket unit according to the invention comprising three components offers an advantageous assembly because these components allow an automated, or at least semi-automated, assembly which simultaneously results in an advantage in costs.
- 20 The gasket unit according to the invention includes a circularly formed, largely dimensioned intermediate space located between the main seal and the front seal. This space can advantageously be used as a reservoir for lubricants and allows lubrication of the journal bearing for life.
- 25 Further additional embodiments of the invention are the subject of dependent claims 2 through 14.

In an advantageous manner the reinforcement of the main seal is engaged with its end in an annular groove of the bearing bushing in order to achieve a 30 defined position. In the area of the cylindrical section of the reinforcement it

is suggested alternatively to provide a circumferential penetration and/or circumferentially distributed, radially outward facing burls, which engage an annular gap at the interior wall of the bearing bushing in a form-fitting manner.

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Furthermore, the reinforcement of the main seal according to the invention is provided at its end facing the rolls with a radially inwardly facing circumferential flange, which is coated by the elastic seal material of the main seal at least at the side of the rolls. Further, the two sealing lips
10 supported on the bearing journal are connected to the elastic seal material enclosing the free end of the inwardly facing flange. The seal material covering the flange of the reinforcement on the outside additionally extends at its face radially beyond the sleeve surface of the cylindrical section of the reinforcement. This measure causes to seal a sealing gap developing between
15 the reinforcement and the interior wall of the bearing bushing. The amount of protruding material, resulting in the difference between the exterior diameter of the reinforcement and the exterior diameter of the flange, coated by the seal material at the exterior, causes a desired friction-locked contact of the seal material at the interior wall of the bearing bushing in the mounted
20 state.

The main seal according to the invention comprises two differently designed sealing lips. The first one, facing the front seal, at its exterior enclosed by the tubular spring, is provided with a triangular profile with the sealing lip that,
25 in the mounted state, is supported on the outer surface of the journal at the triangular tip. The other sealing lip has a primarily rectangularly designed profile. Based on the angled arrangement of the sealing lip in the direction of the opening of the bearing bushing, the sealing lip is supported on the journal via an edge support.

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The mounted state of the sealing lips, i.e. the largely symmetrical arrangement of the triangular frontal sealing lip and the diagonally arranged additional sealing lip causes a diagonally extending gap between the sealing lips with a rounded gap end. The gap allows the desired high elasticity
5 and/or the independent support of the sealing lips on the sleeve surface of the journal. Furthermore, this gap simultaneously forms another desired reservoir for the lubricant of the journal bearing.

The front seal according to the invention, made exclusively from an elastic
10 material is arranged on the journal in a torque-proof manner. For this purpose, the journal is advantageously cascaded so that the front seal is positioned at a section of the journal that has a greater diameter than the section on which the sealing lips of the main seal are supported.

15 The design of the front seal comprises an axially designed, U-shaped part, which the radially cascading end section of the bearing bushing engages. The seat of the front seal is limited on the outside by a rim and on the inside by the sealing lip of the front seal.

20 In order to form a labyrinth seal, the rim of the front seal forms on its end a radially inwardly facing projection, which in the mounted state engages in a form-fitting manner an annular gap of the bearing bushing, which is radially cascading in its end section.

25 Another embodiment of the front seal provides for the sealing lip to be supported in a form-fitting manner at the reinforcement of the main seal in the mounted state. Due to the fact that the section of the reinforcement, with its end facing radially outward, forms a conically shaped integrated diagonal area, the assembly of the front seal is facilitated. The integrated diagonal
30 area allows the sealing lip to assume the mounting position without any

trouble. The particular purpose of the sealing lip of the front seal is to prevent negative influences on the journal bearing, such as the entry of contaminants or moisture into the interior chamber of the bearing bushing.

- 5 For bearing bushings to be provided with secondary lubrication it is suggested to provide the sealing lip with at least one front groove, axially extending in the area of the contact zone. If necessary, the front seal may be provided with several grooves distributed over the circumference, in order to avoid a closed, line-contacting support of the front seal on the reinforcement
10 of the main seal.

The embodiment of the front seal, provided with a circular shape, further provides that it includes an axial rim on a side facing away from the main seal, in the mounted position of the device supported in a friction-locked
15 manner on a shoulder of the journal. This measure improves the seal of the front seal in reference to the journal and thus avoids any discharge of lubricants from the bearing bushing and/or the entry of any contaminants via the sealing gap developing between the front seal and the journal. In order to achieve a spatially optimized design of the front seal, said seal is provided
20 with an exterior diameter equivalent to or smaller than an exterior diameter of the bearing bushing.

BRIEF DESCRIPTION OF THE DRAWING

- 25 The invention is explained in the following detailed description on the basis of a single figure showing an exemplary embodiment of said invention.

DETAILED DESCRIPTION OF THE DRAWING

The figure shows the construction of a gasket unit 1 according to the invention provided for a journal bearing 2. Here, a bearing bushing 3 is provided, in which a journal 4 is supported via rollers 5. The bearing bushing 3, provided in a cup-shaped design, extends axially over the rollers 5 and forms an annular gap 6 limited radially outwardly by an interior wall 7 of the bearing bushing 3 and radially inwardly by the journal 4. The gasket unit 1, which serves to seal the annular gap 6, comprises the components of the main seal 8, the front seal 9, and the spring washer 10. In the mounted state the main seal 8 is formed by a reinforcement 11, with its cylindrical section being supported at the interior wall 7 of the bearing bushing 3 in a friction-locked manner. At the end facing the front seal 9 the reinforcement 11 is provided with a radially outwardly facing section 12, engaged in a annular gap 13 of the bearing bushing 3 in a form-fitting manner and, thus ensuring a defined positioning of the main seal 8.

A predetermined axial front tension of the rollers 5 is caused by the spring washer 10, which is supported on the inside on a face 23 of the rollers 5 and on the outside on the seal material 15 and, thus, indirectly on the flange 14 of the reinforcement 11. At the end facing the spring washer 10 the reinforcement 11 forms a radially inwardly facing flange 14, which is coated with an elastic seal material 15 over its entire surface on the side facing the spring washer. Here, the seal material 15 extends, limited at the outside radially via the exterior diameter of the reinforcement 11, and thus causes a sealing gap 16 between the interior wall 7 and the cylindrical section 17 of the reinforcement 11. At the free end of the flange 14, the main seal 8 is provided with two axially spaced apart sealing lips 18, 19 contacting a section 20 having the same diameter as the journal 4 in a sealing and friction-locked manner. The sealing lip 19 at the end has a triangular cross-sectional profile,

which in the mounted position is supported largely symmetrically oriented contacting the section 20 of the journal 4 at a line. In order to achieve an enhanced contact pressure, the sealing lip 19 is enclosed on the outside by a tubular spring 21, which is inserted into an annular gap 22 of the sealing lip
5 19. The other sealing lip 18 is oriented at an angle diagonally starting from the flange 14 in a direction of the symmetry axis of the journal 4, and forms a rectangular cross-sectional profile, supported with its exterior edge on the section 20 of the journal 4.

10 Between the sealing lips 18, 19 a diagonally extending groove 24 is formed, having a rounded end. The groove 24 improves, on the one hand, a desired elasticity of the sealing lips 18, 19 and thus any separate effectiveness. Furthermore, the groove 24 can be used as a reservoir for lubricants for the journal bearing 2.

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The front seal 9 is made exclusively from a seal material and is arranged in a friction-locked manner on the section 25 of the journal 4, in an area where its diameter exceeds the diameter of the section 20. The front seal 9 forms an axially oriented recess 26 facing in a direction of the bearing bushing 3.

20 Radially on the outside, the recess 26 is limited by the rim 27 and on the inside by the sealing lip 28. An end section 29 of the bearing bushing 3 is fitted into the recess 26 of the front seal 9, with the rim 27 of the front seal 9 contacting an exterior surface of the end section 29 in a non-positive manner. In order to form a labyrinth seal 30 between the end section 29 and the rim
25 27 of the front seal 9, on the inside of the free end of the rim 27 a radially inwardly facing projection 31 is provided, which engages a circumferential groove 32 of the end section 29.

The sealing lip 28 of the front seal 9 is allocated to the reinforcement 11 of
30 the main seal 8. In order to facilitate later lubrication of the journal bearing

2, the sealing lip 28 is provided with at least one axially extending groove 37 in the area of the contact zone 36, allowing a simplified entry of the lubricant into a space 35 located axially between the front seal 9 and the main seal 8, which can be used as a reservoir for lubricants. The section 12 of the
5 reinforcement 11, extending with its end radially outward at an angle, allows a destruction-free assembly of the front seal 9 and the corresponding sealing lip 28. The front seal 9 also comprises a rim 33 on the side opposite the main seal 8, which in the mounted state is supported on a shoulder 34 of the journal 4 in a friction-locked manner and, thus, sealing.

List of Reference Characters

	1	Gasket unit
	2	Journal bearing
5	3	Bearing bushing
	4	Journal
	5	Roller
	6	Annular gap
	7	Interior wall
10	8	Main seal
	9	Front seal
	10	Spring washer
	11	Reinforcement
	12	Section
15	13	Annular gap
	14	Flange
	15	Seal material
	16	Sealing gap
	17	Section
20	18	Sealing lip
	19	Sealing lip
	20	Section
	21	Tubular spring
	22	Annular gap
25	23	Face
	24	Groove
	25	Section
	26	Recess
	27	Rim
30	28	Sealing lip

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- 29 End section
- 30 Labyrinth seal
- 31 Projection
- 32 Circumferential groove
- 5 33 Rim
- 34 Shoulder